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- USPC ..... 42/73, 74  
See application file for complete search history.

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- (\*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

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*F41C 23/20* (2006.01)  
 (Continued)

- (52) **U.S. Cl.**  
CPC ..... *F41C 23/08* (2013.01); *F41C 23/06*  
(2013.01); *F41C 23/14* (2013.01); *F41C 23/20*  
(2013.01)

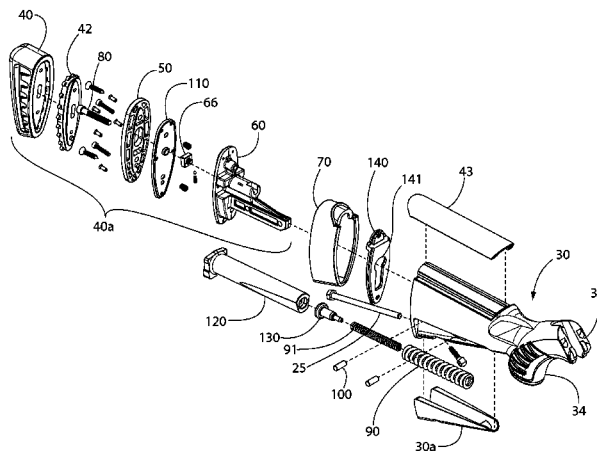
- (58) **Field of Classification Search**  
CPC ..... F41A 23/20; F41A 23/00; F41A 23/08;  
F41A 23/14; F41A 23/06; F41C 23/20;  
F41C 23/00; F41C 23/08; F41C 23/14;  
F41C 23/06

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- (57) **ABSTRACT**

A user-adjustable recoil reduction system for a firearm includes a buttstock and a butt pad assembly movably mounted to the buttstock. A spring is disposed in the buttstock which is acted on by opposing front and rear spring guides. A preload adjustment screw is rotatably coupled to the butt pad assembly which engages the rear spring guide. Rotating the screw in opposing directions alternately advances or retracts the rear spring guide in the buttstock. Advancing the rear spring guide compresses the spring to set a first spring preload condition and retracting the rear spring guide relaxes the spring to set a second spring preload condition different than the first condition. The spring preload conditions may be selected to match firing light or heavy type ammunition shell loads. In one embodiment, the adjustment screw is accessible for adjusting the preload without removing the butt pad assembly from the buttstock.

**23 Claims, 18 Drawing Sheets**



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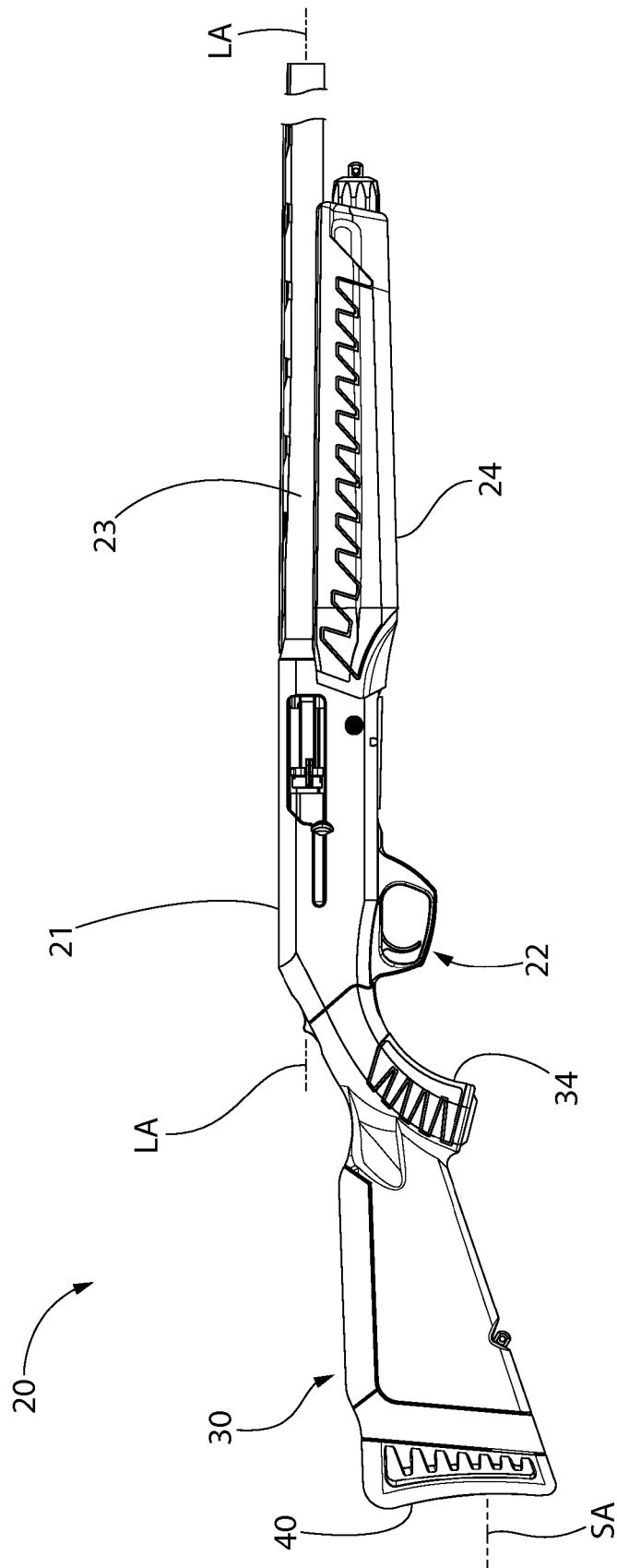


FIG. 1

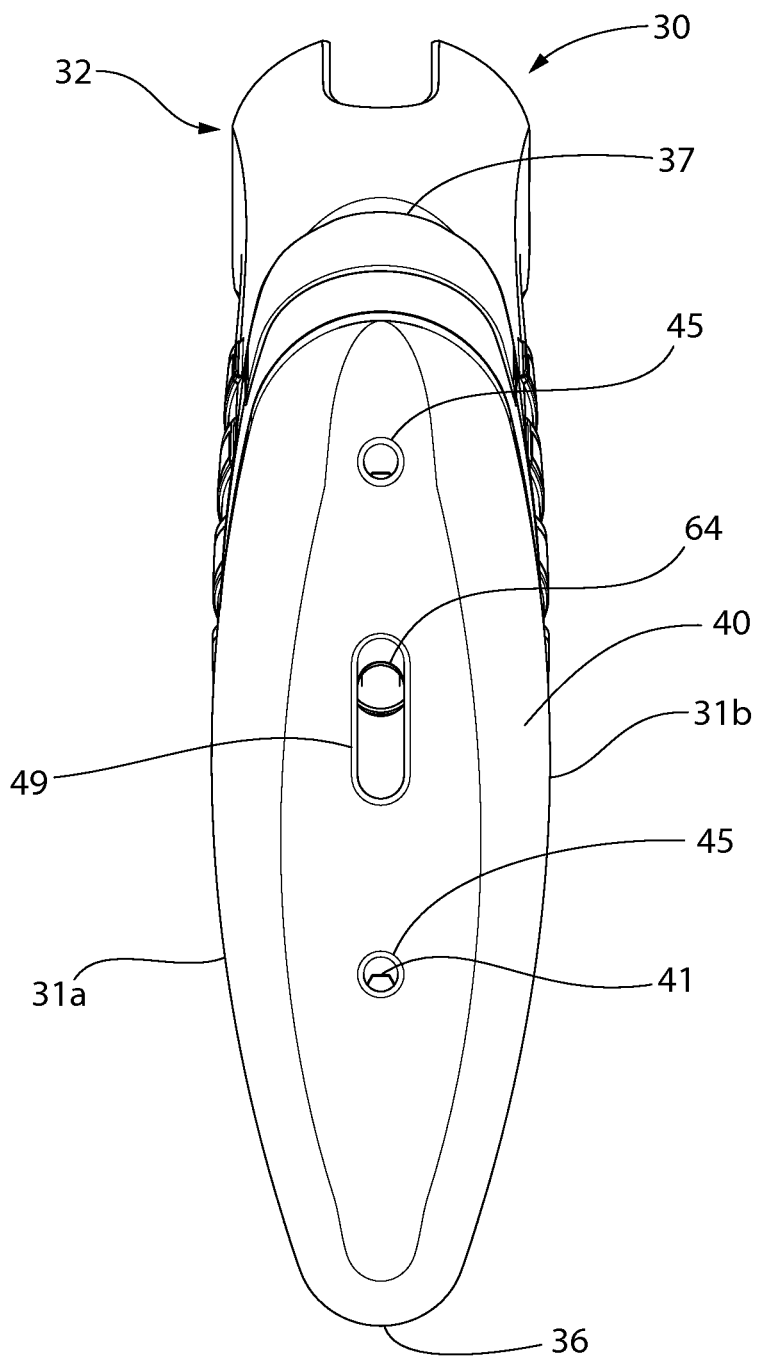


FIG. 2

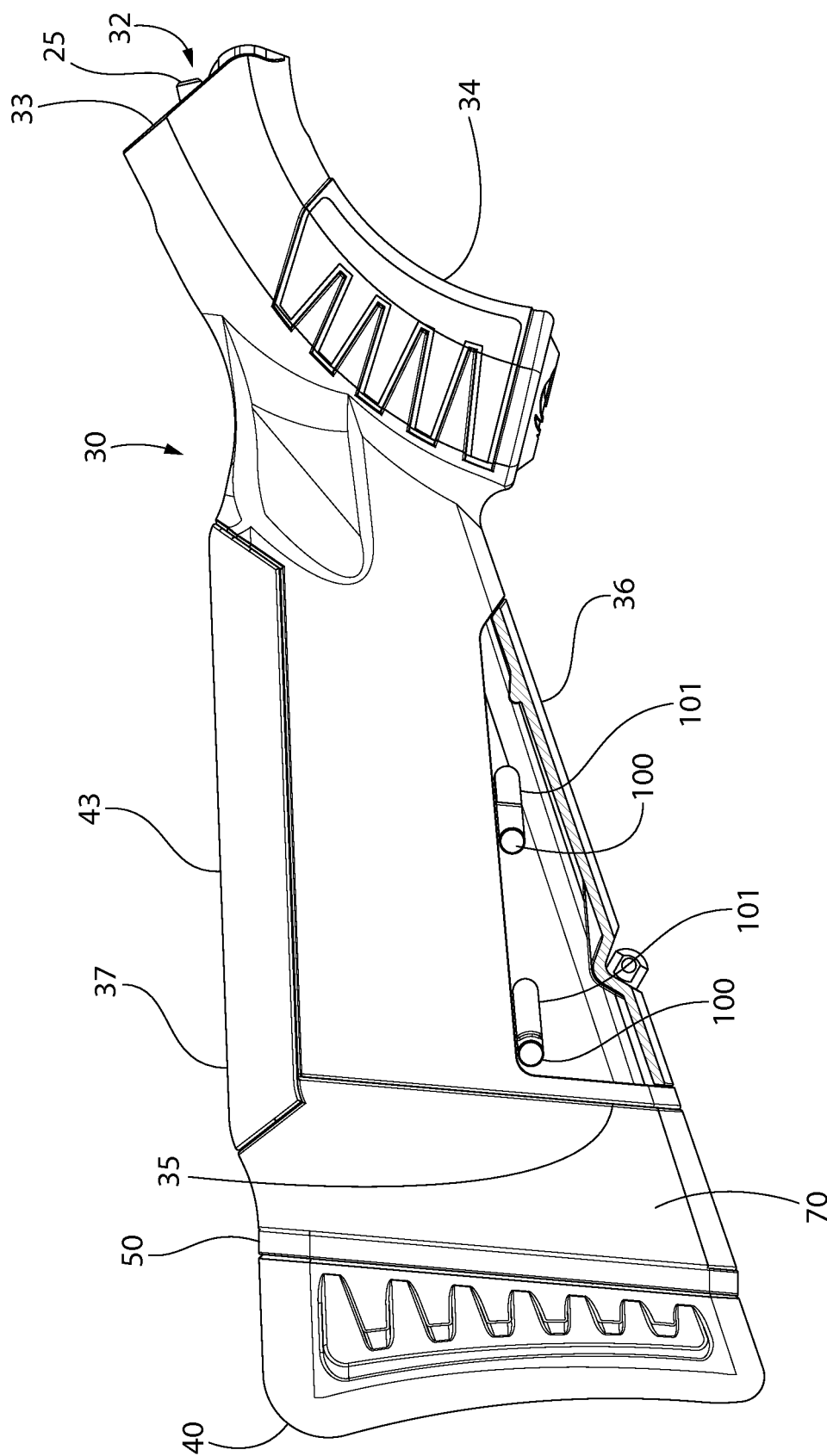


FIG. 3

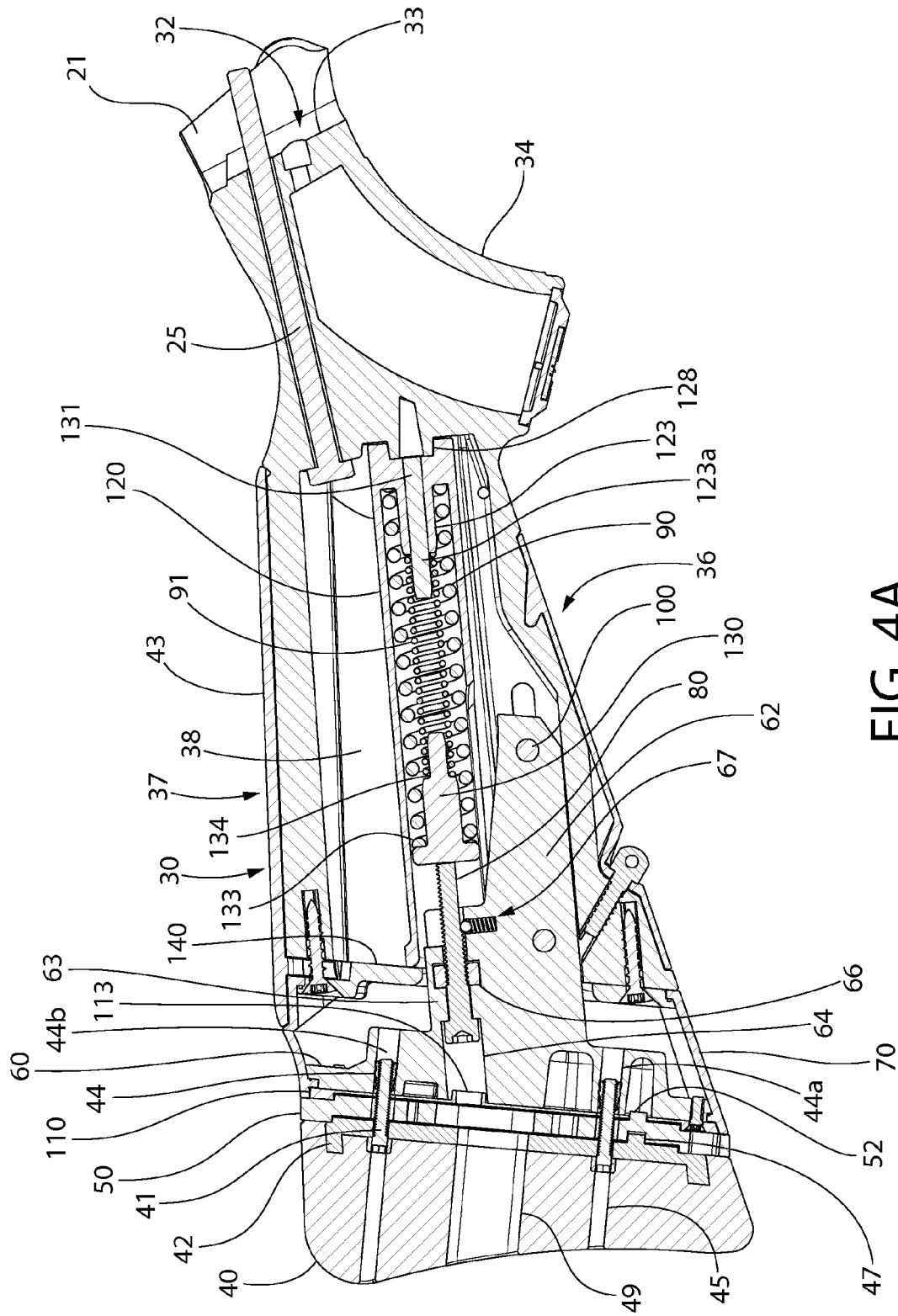


FIG. 4A

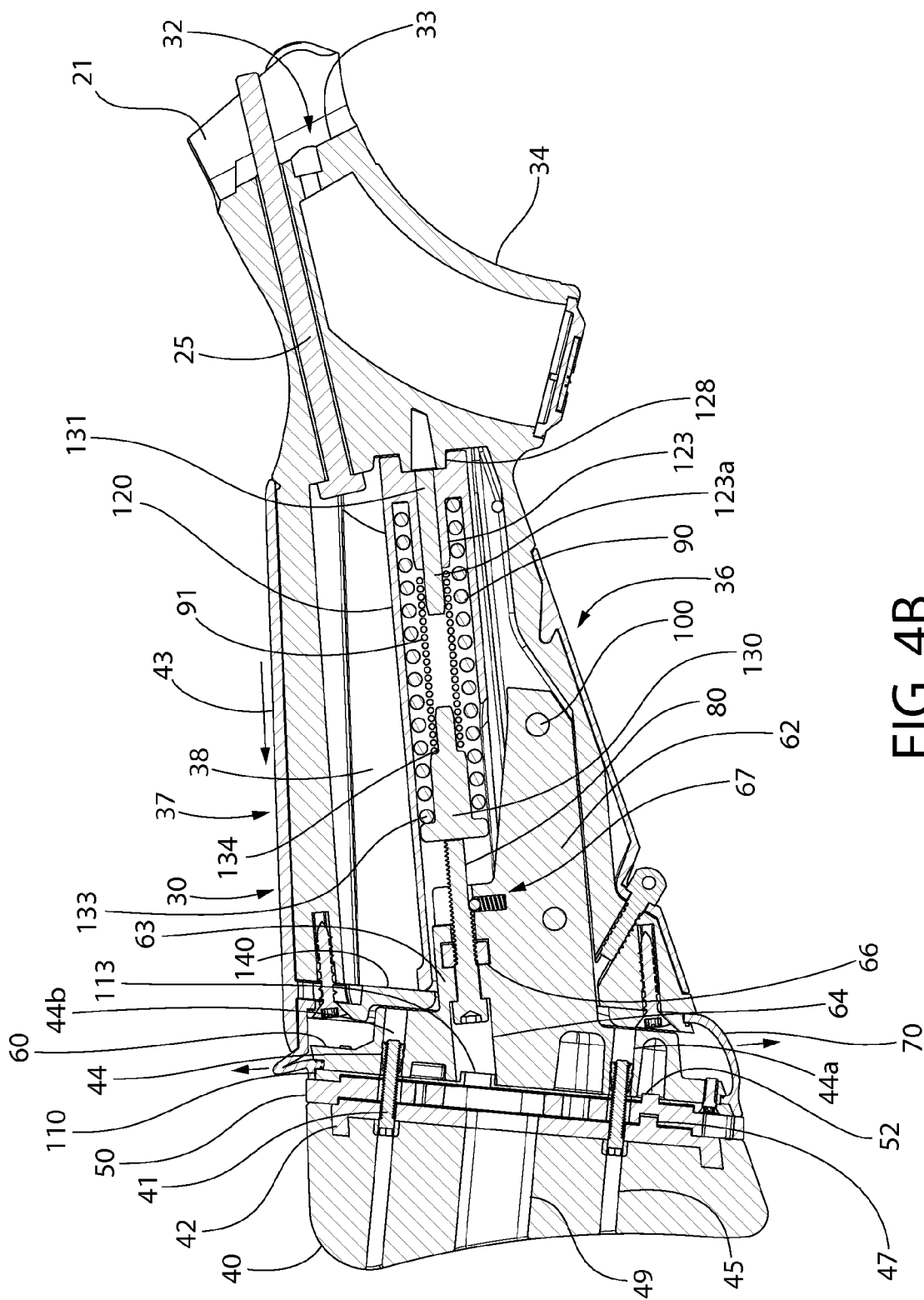


FIG. 4B

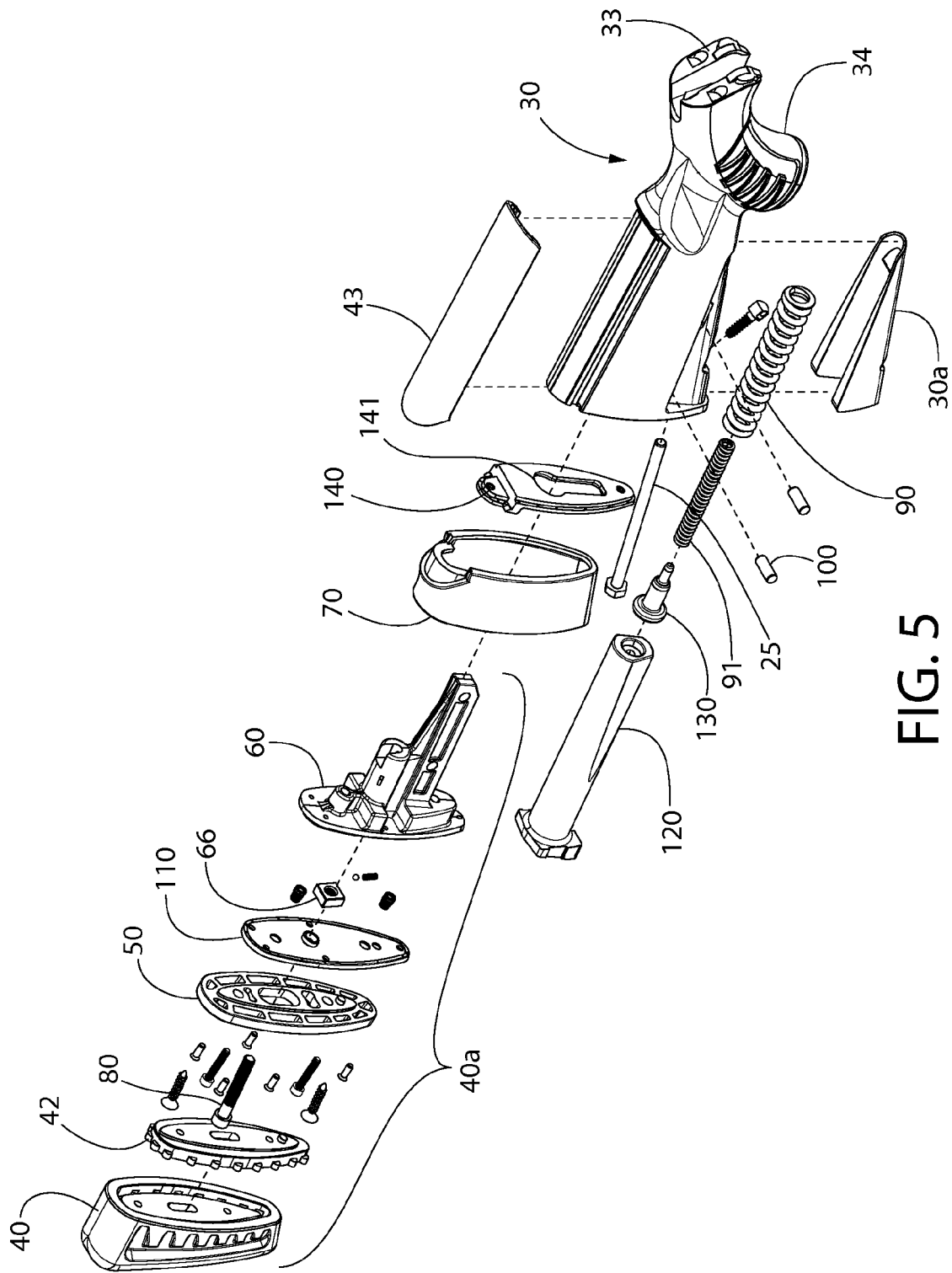


FIG. 5

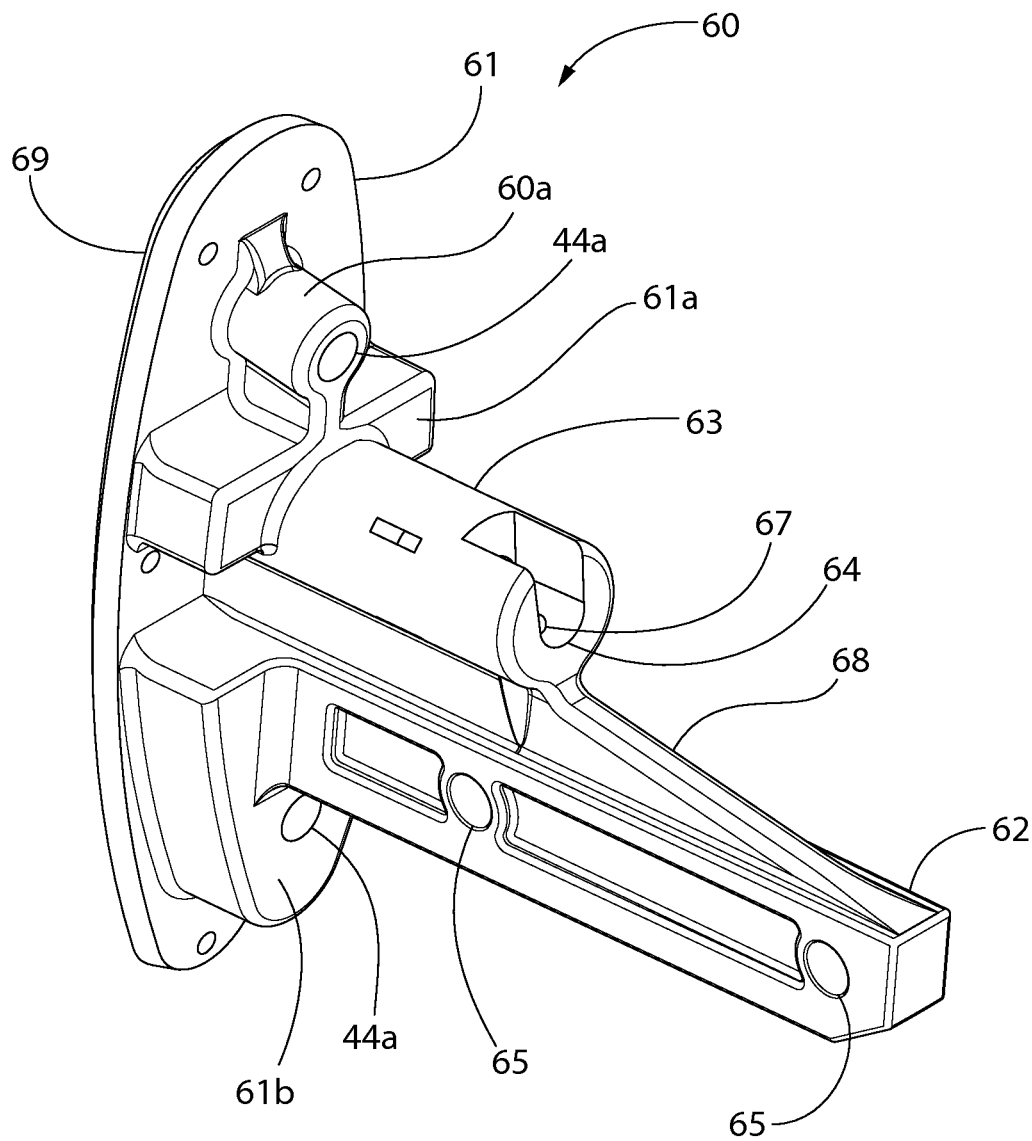


FIG. 6A

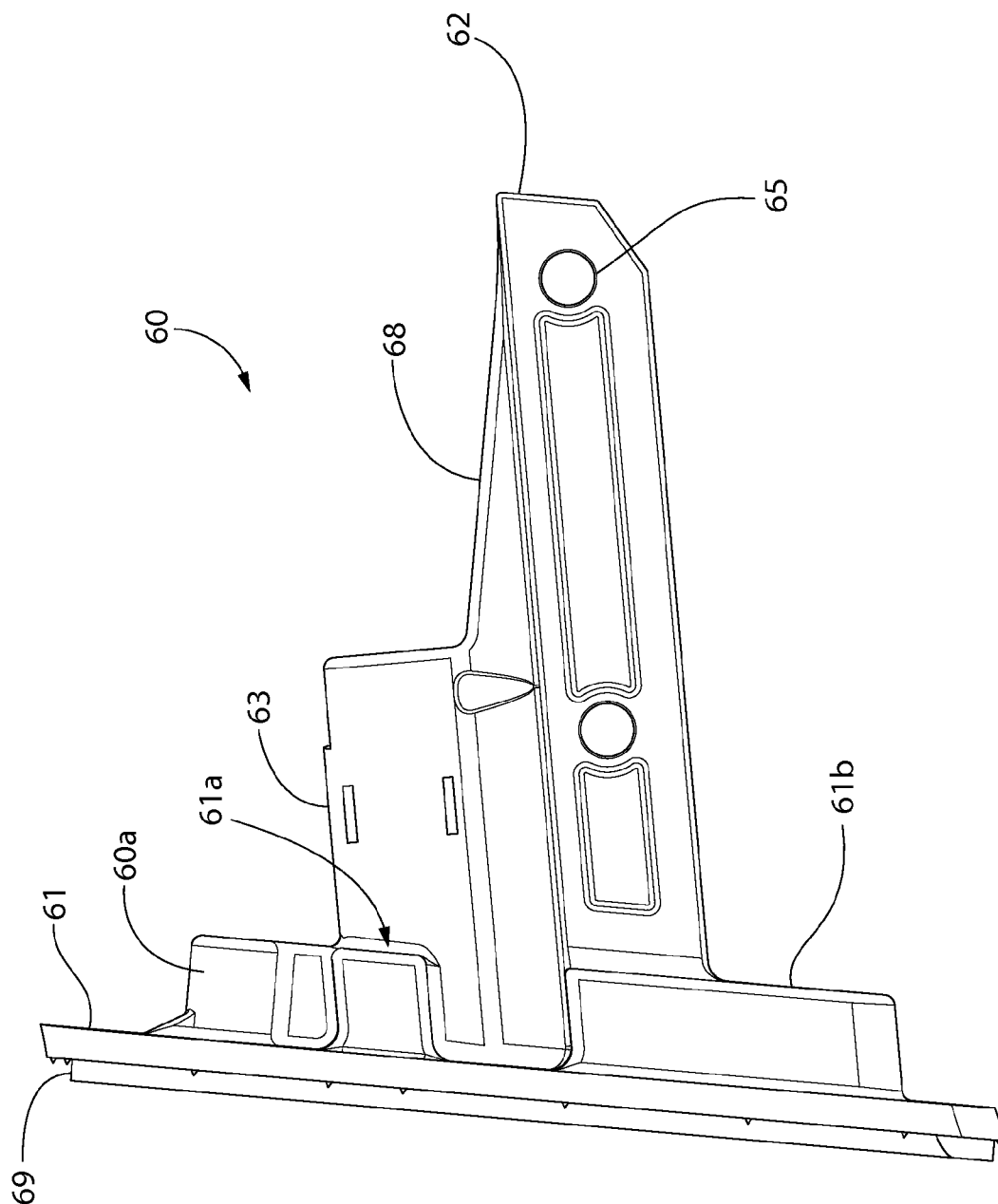


FIG. 6B

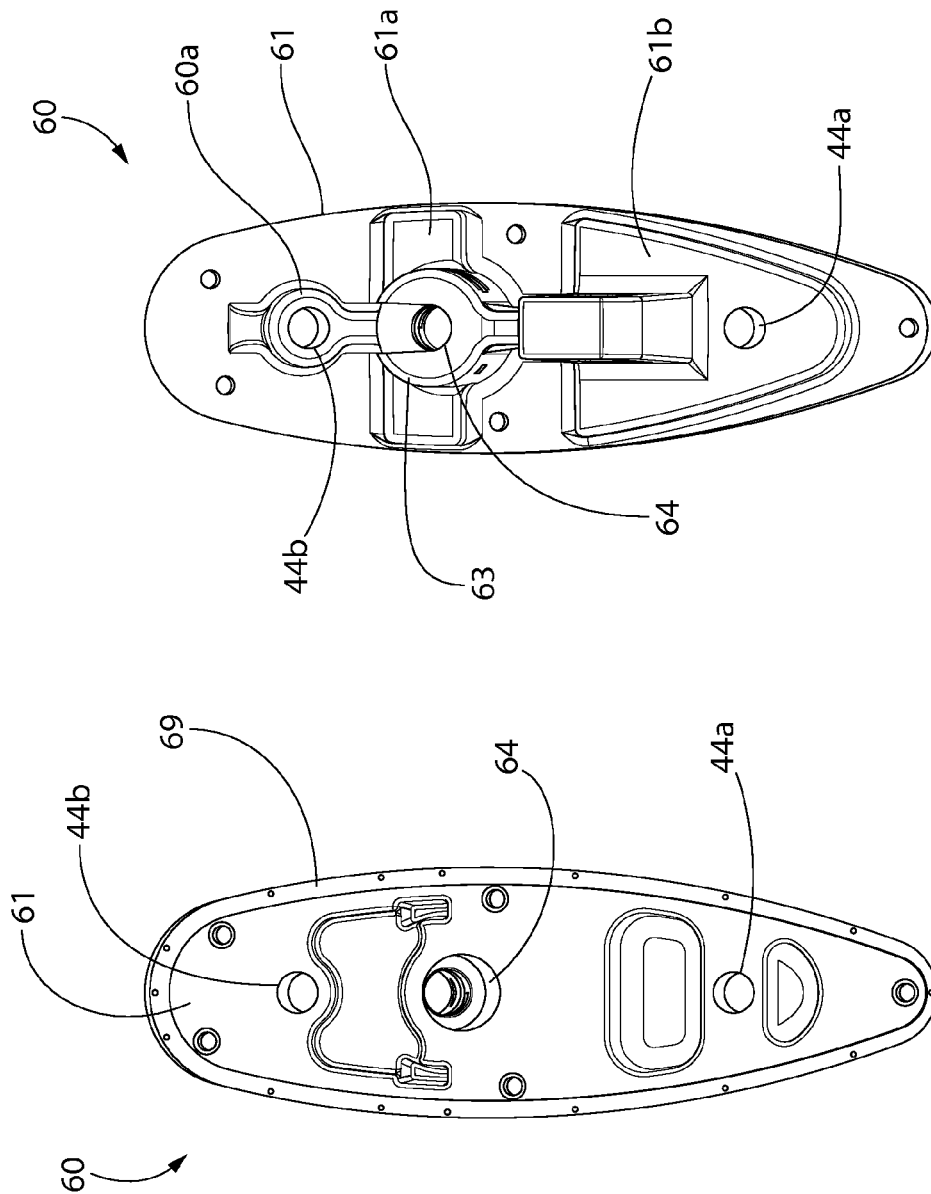


FIG. 6D

FIG. 6C

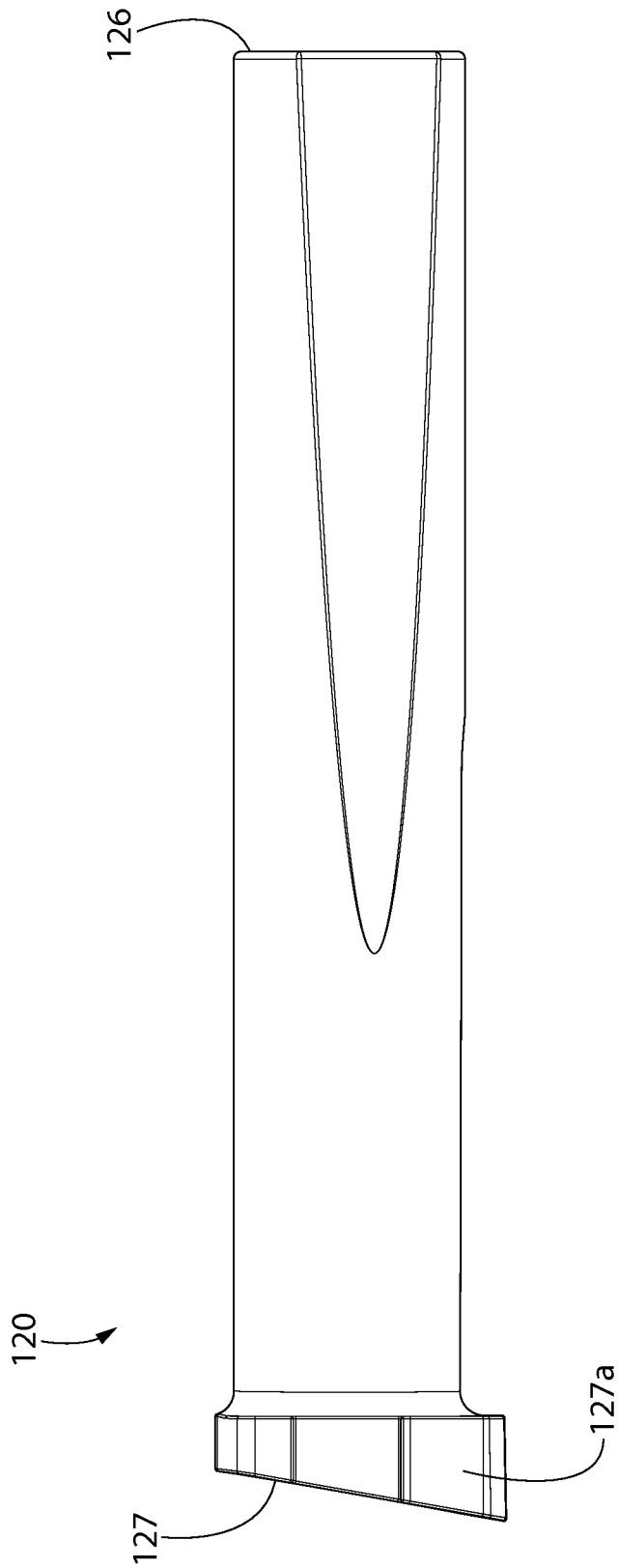


FIG. 7A

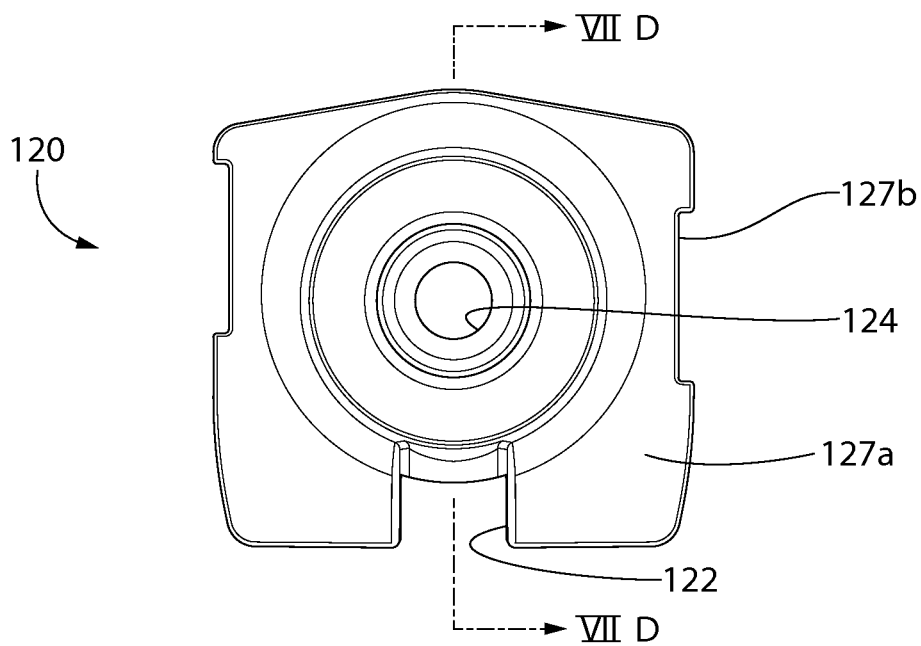


FIG. 7B

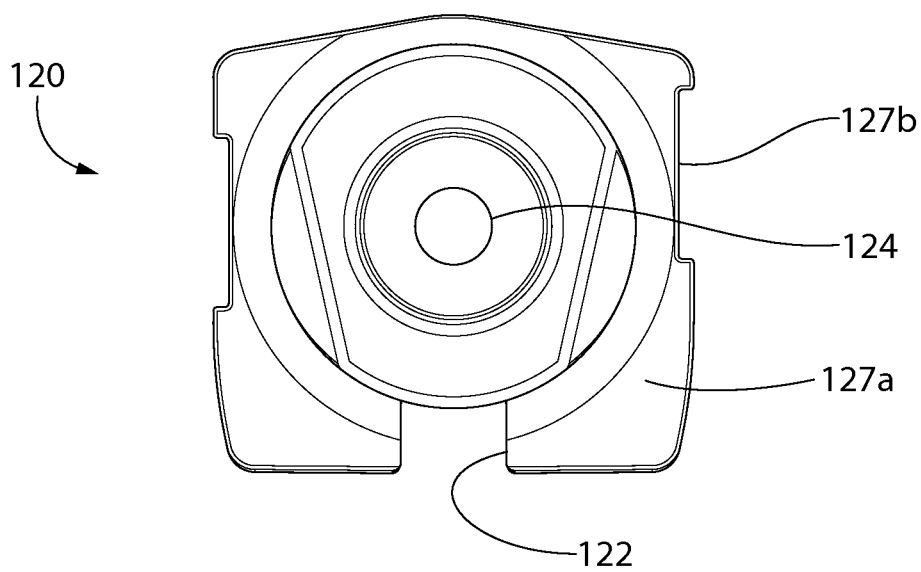


FIG. 7C

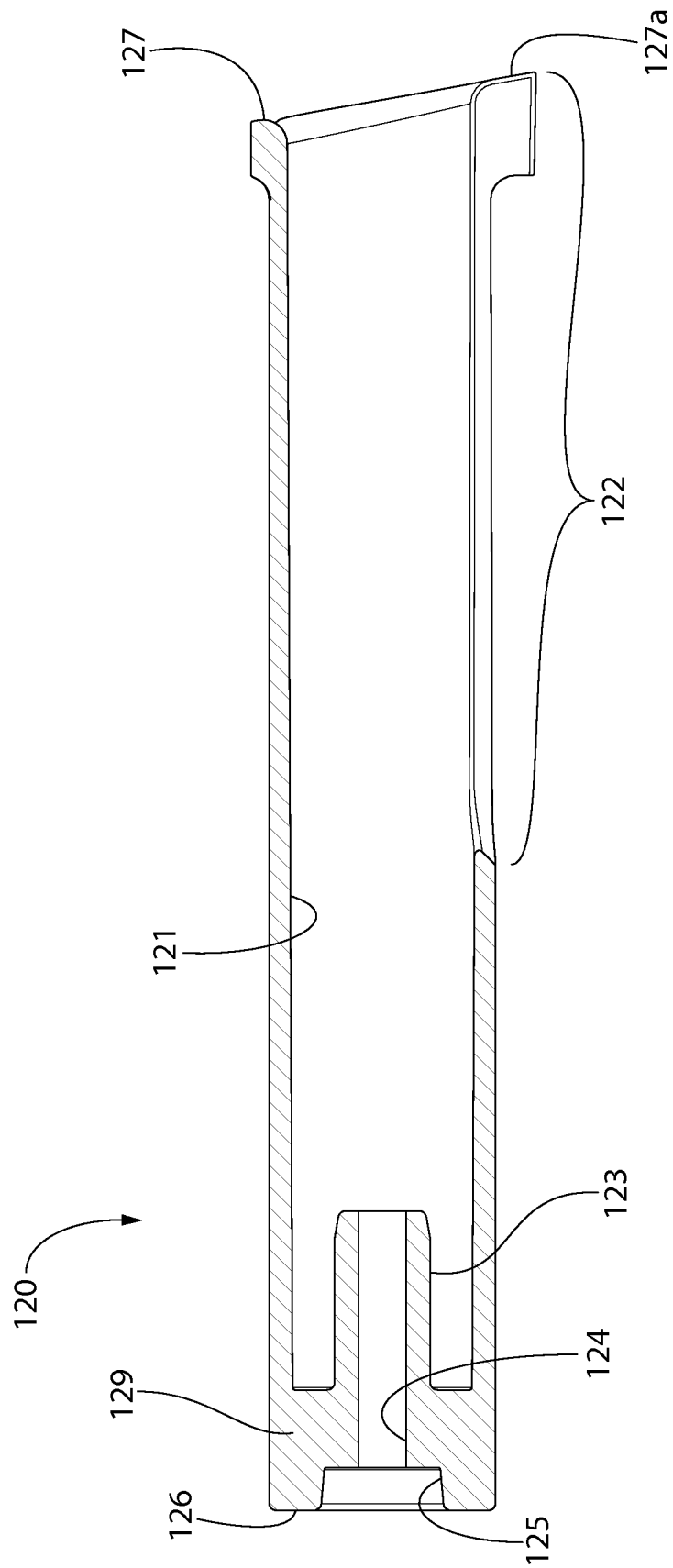


FIG. 7D

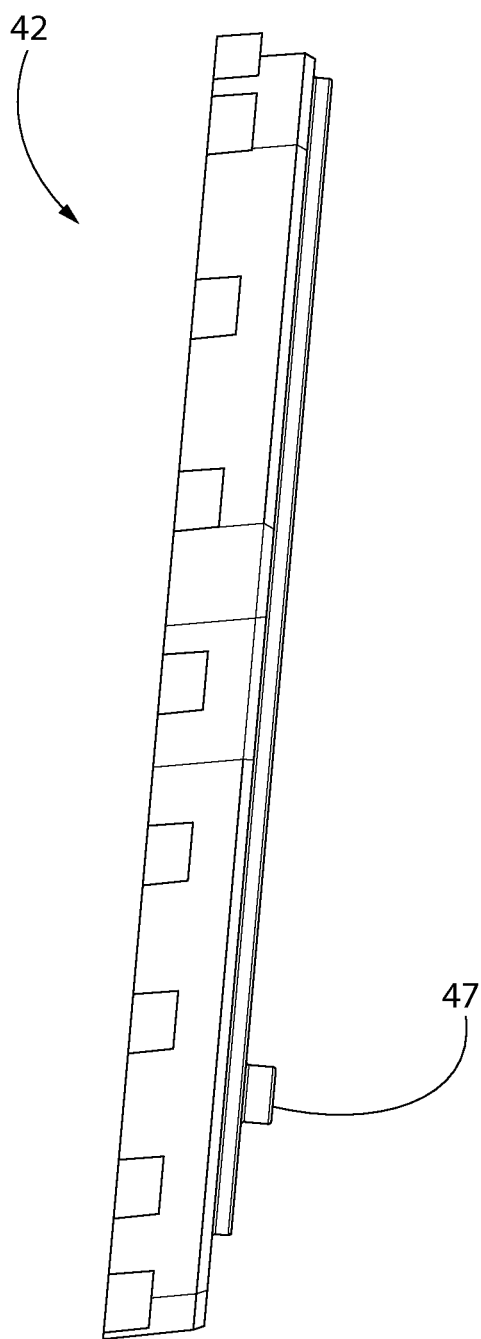


FIG. 8A

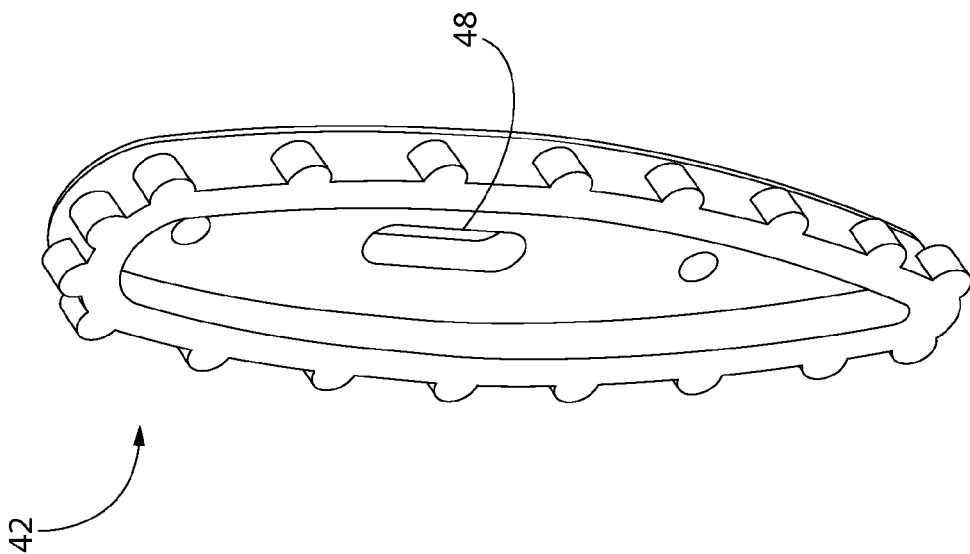


FIG. 8B

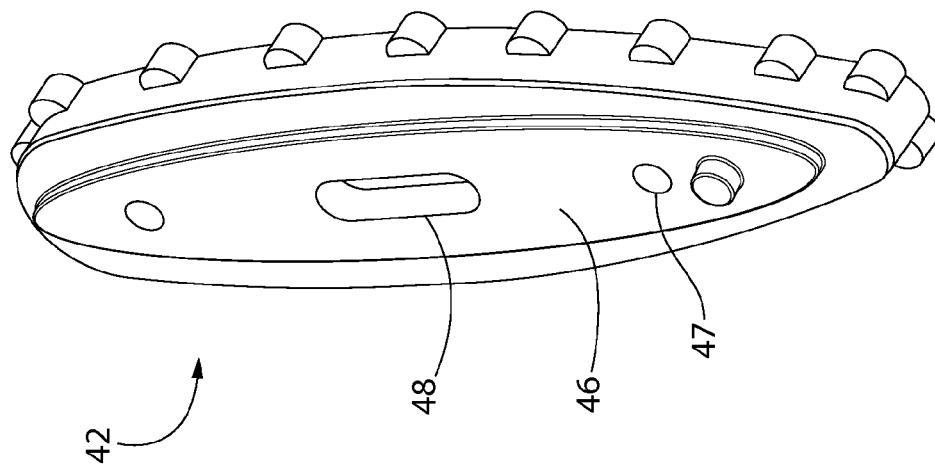


FIG. 8C

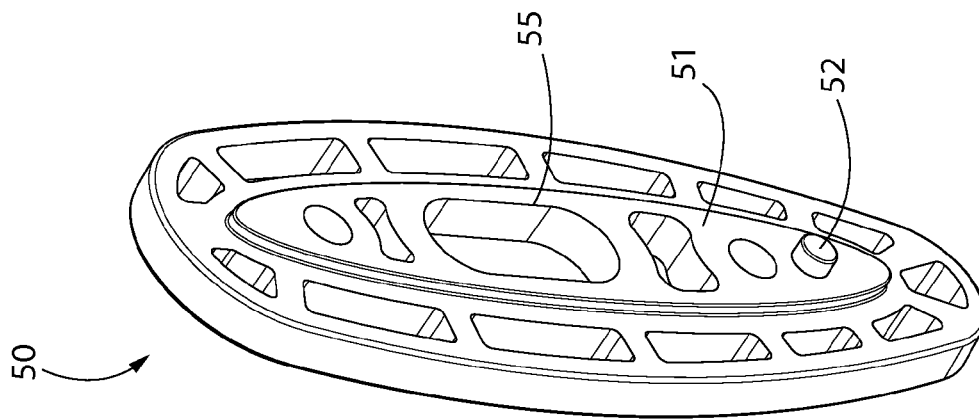


FIG. 9A

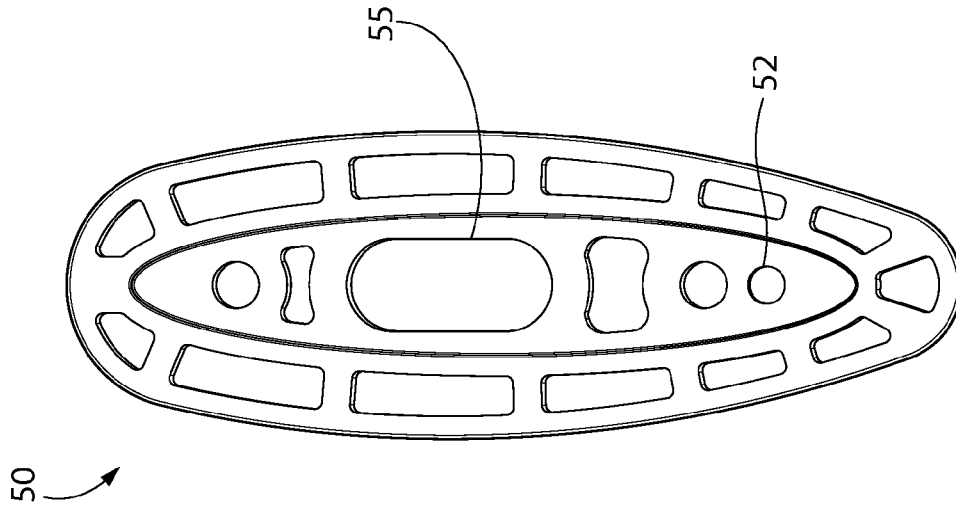


FIG. 9B

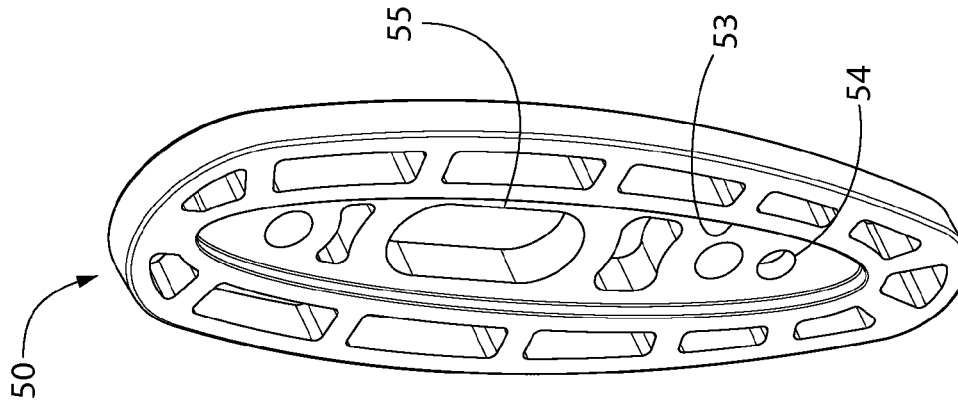


FIG. 9C

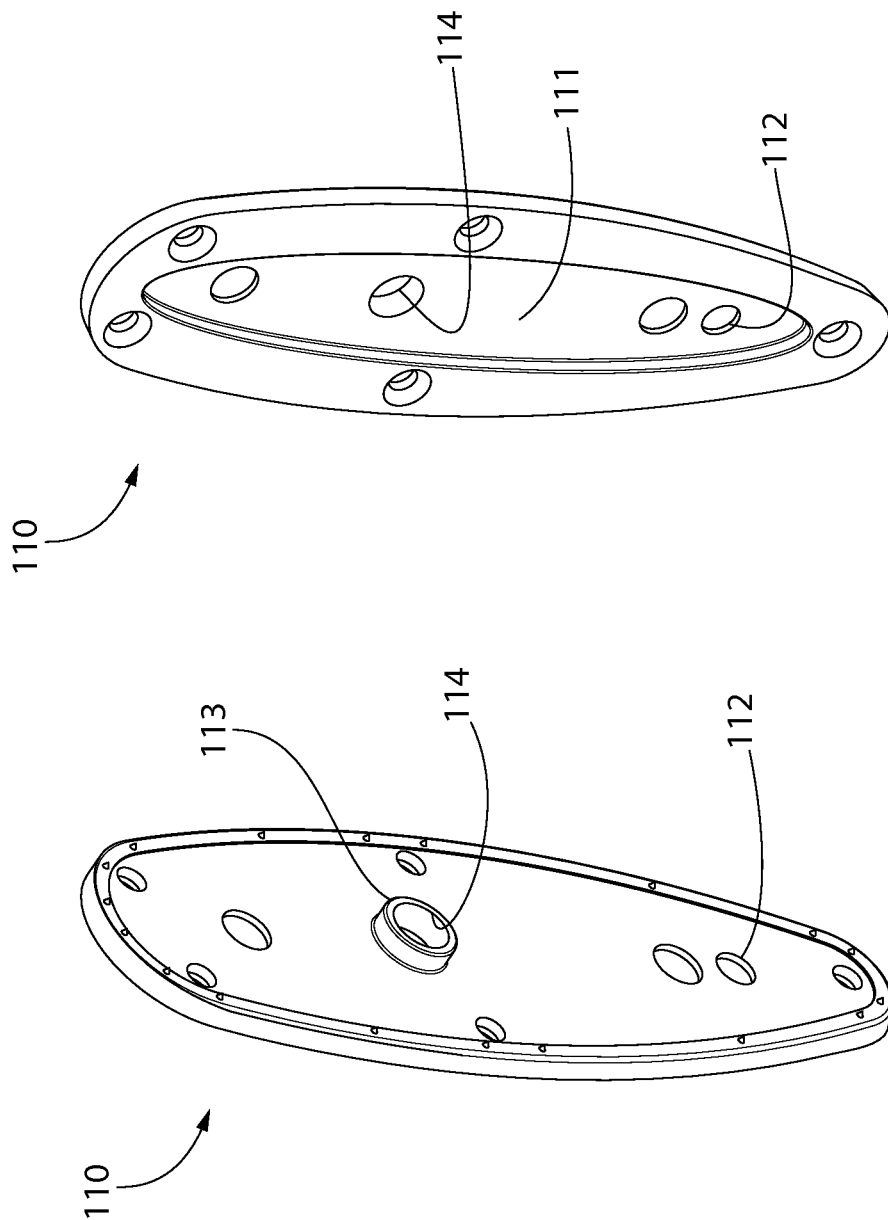


FIG. 10B

FIG. 10A

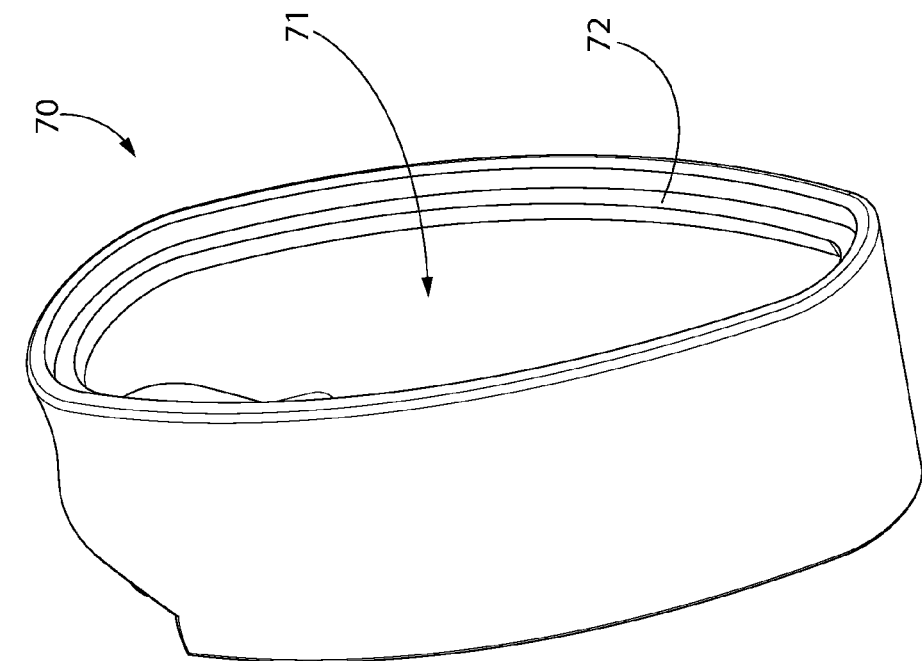


FIG. 11B

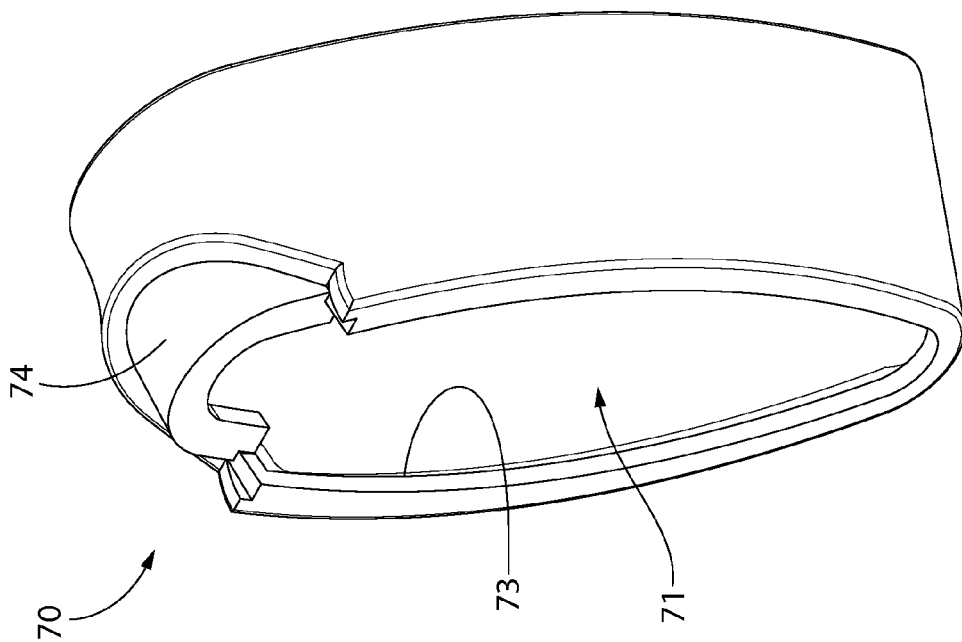


FIG. 11A

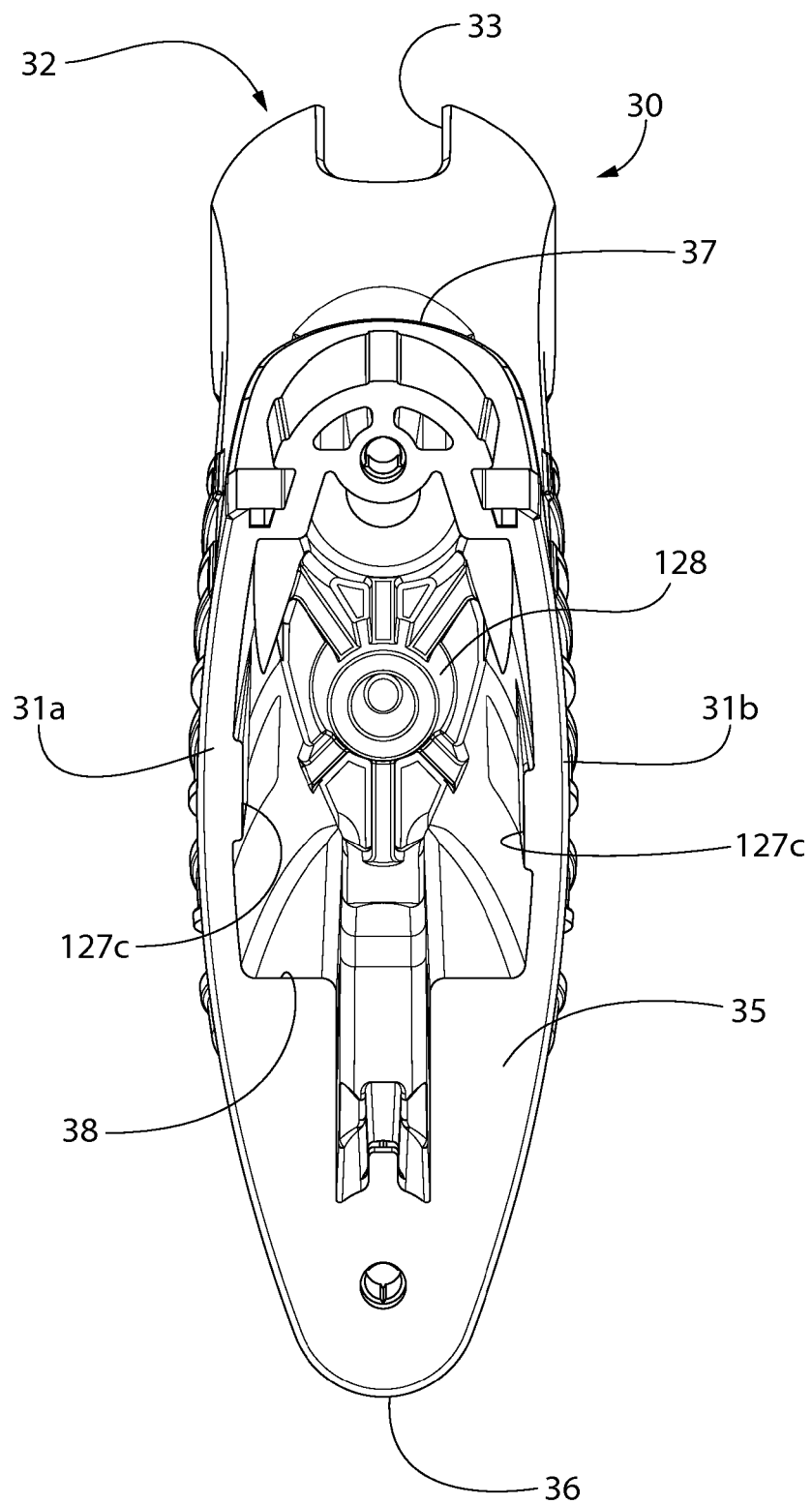


FIG. 12

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## RECOIL REDUCTION SYSTEM FOR FIREARM

### CROSS-REFERENCE TO RELATED APPLICATIONS

The present application claims the benefit of priority to U.S. Provisional Application No. 62/022,461 filed Jul. 9, 2014, the entirety of which is incorporated herein by reference.

### BACKGROUND

The present invention generally relates to firearms, and more particularly to systems for reducing felt recoil from discharging the firearm.

A recoil force is generated when a firearm is discharged. The bullet or slug and high pressure combustion gases generated exit the muzzle end of the barrel in a forward direction at considerable velocity. This creates a recoil force which drives the firearm in an opposite rearward direction towards the shooter (user) under the principles of momentum. The recoil force generated is substantially equal to the forward discharge force of the propellant gases. Various firearm recoil reduction approaches have been used.

When firing a long gun such as a rifle or shotgun which may use somewhat high power ammunition, the felt recoil may especially be significant and uncomfortable for the user. In addition, this may make it more difficult to reacquire a target and fire a second shot accurately. In the case of a shotgun, a user may also sometimes find it desirable to alternate between firing light load shotshells on some occasions and heavy load shotshells on other occasions depending on the type of target shooting and/or hunting activity planned. This also is applicable to switching between different cartridges in a rifle for firearms provided with this capability.

Accordingly, it is therefore desirable to at least partially abate and lessen the recoil forces acting against the user's arms and shoulder against which the buttstock of a rifle or shotgun is typically abutted. It is also further desirable to have the ability to adjust a recoil adjustment system for a shotgun or rifle to accommodate firing different type loads.

### SUMMARY

A recoil reduction system is provided for a firearm which may be mounted in the buttstock of a rifle or shotgun. The system includes a spring suppression mechanism to dampen felt recoil. In one embodiment, the recoil reduction system includes a user-tunable adjustment feature allowing the suppression or dampening capacity of the system to be changed to accommodate firing different types of loads.

According to an aspect of the invention, an adjustable recoil reduction system for a firearm includes: an axially extending buttstock having a longitudinal stock axis, a rear end, a front end, and a longitudinally extending internal cavity; a ram including a cantilevered mounting extension, the mounting extension projecting forward from the ram through the rear end of the buttstock into the internal cavity, the mounting extension slideably coupling the ram to the buttstock; a butt pad coupled to the ram; an adjustable plunger assembly comprising a rear spring guide disposed in the buttstock and a preload adjustment screw rotatably coupled to the ram and engaging the rear spring guide, the plunger assembly being movable forward and rearward in the buttstock; and a first compression spring mounted inside

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the buttstock and engaged by the rear spring guide, the plunger assembly operating to compress the first compression spring when the buttstock moves rearward relative to the plunger assembly; wherein rotating the adjustment screw in a first direction advances the rear spring guide and compresses the first compression spring for setting a first preload condition, and rotating the adjustment screw in an opposite second direction retracts the rear spring guide and allows the first compression spring to expand for setting a second preload condition; wherein when a rearward acting recoil force is generated by discharging the firearm, the buttstock moves rearward and compresses the first compression spring against the plunger assembly to absorb at least a portion of the recoil force.

According to another aspect of the invention, an adjustable recoil reduction system for a firearm includes: a buttstock extending rearward from a receiver, the buttstock having a longitudinal stock axis, a rear end, a front end, and an internal cavity extending between the front and rear ends; a spring assembly disposed in the buttstock, the spring assembly comprising a tubular sleeve fixedly mounted inside the buttstock, a first spring inside the sleeve, a second spring inside the sleeve concentrically arranged around the first spring, a rear spring guide engaged with the first and second springs, and a front spring guide engaged with the first and second springs; a butt pad assembly comprising a butt pad configured for placement against a shoulder of a user and a ram coupled to butt pad, the butt pad assembly movably coupled to the buttstock; an axially elongated preload adjustment screw threadably coupled to the ram and having a front end engaging the rear spring guide to fix a position of the rear spring guide relative to the sleeve, the preload adjustment screw movable forward and rearward relative to the buttstock by rotating the preload adjustment screw in opposing directions; wherein rotating the preload adjustment screw in a first direction compresses the first and second springs to set a first preload condition, and rotating the preload adjustment screw in an opposite second direction expands the first and second springs to set a second preload condition; wherein when a rearward acting recoil force is generated by discharging the firearm, the buttstock moves rearward and the rear spring guide compresses the first and second springs thereby absorbing at least a portion of the recoil force.

A method for reducing recoil in a firearm includes: providing a firearm including a buttstock, a butt pad assembly movably coupled to a rear end of the buttstock, a coiled first spring disposed inside the buttstock, and an adjustable plunger mechanism comprising a rear spring guide engaging the compression spring and a preload adjustment screw rotatably coupled to the butt pad assembly and engaging the rear spring guide, the buttstock being in a forward non-recoil position spaced apart from the butt pad assembly by a first distance; rotating the preload adjustment screw in a first direction which advances the rear spring guide forward in the buttstock; compressing the first spring by the advancement of the rear spring guide to a first preload condition; discharging the firearm; moving the buttstock in a rearward direction under recoil closer to the butt pad, the buttstock being in a rearward recoil position spaced apart from the butt pad assembly by a second distance smaller than the first distance; the buttstock compressing the first spring by movement in the rearward direction; expanding the first spring; and returning the buttstock to the forward non-recoil position.

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Further areas of applicability of the present invention will become apparent from the detailed description provided hereinafter.

#### BRIEF DESCRIPTION OF THE DRAWINGS

The features of the exemplary embodiments will be described with reference to the following drawings where like elements are labeled similarly, and in which:

FIG. 1 is a longitudinal side view of one exemplary embodiment of a firearm including an adjustable recoil mechanism according to the present disclosure;

FIG. 2 is a rear end view thereof;

FIG. 3 is a side elevation view of the buttstock of the firearm of FIG. 1 with detachable bottom cover removed;

FIG. 4A is a cross-sectional view thereof showing the buttstock in a forward non-recoil position;

FIG. 4B is a cross-sectional view thereof showing the buttstock in a rearward recoil position;

FIG. 5 is an exploded perspective view thereof;

FIGS. 6A-D are perspective, side, rear, and front views respectively of the ram shown in FIGS. 4A-B;

FIGS. 7A-D are side, rear, front, and cross-sectional views respectively of the spring tube shown in FIGS. 4A-B;

FIGS. 8A-C are side, rear perspective, and front perspective views respectively of the butt pad hard insert shown in FIGS. 4A-B;

FIGS. 9A-C are front perspective, front, and rear perspective views respectively of the spacer plate shown in FIGS. 4A-B;

FIGS. 10A-B are front and rear perspective views respectively of the skirt retaining plate shown in FIGS. 4A-B;

FIGS. 11A-B are rear and front perspective views respectively of the skirt shown in FIGS. 4A-B; and

FIG. 12 is a rear view of the buttstock with butt pad assembly and skirt mounting plate removed.

All drawings are schematic and not necessarily to scale. Parts given a reference numerical designation in one figure may be considered to be the same parts where they appear in other figures without a numerical designation for brevity unless specifically labeled with a different part number and/or described herein.

#### DETAILED DESCRIPTION

The features and benefits of the invention are illustrated and described herein by reference to exemplary embodiments. This description of exemplary embodiments is intended to be read in connection with the accompanying drawings, which are to be considered part of the entire written description. Accordingly, the disclosure expressly should not be limited to such exemplary embodiments illustrating some possible non-limiting combination of features that may exist alone or in other combinations of features.

In the description of embodiments disclosed herein, any reference to direction or orientation is merely intended for convenience of description and is not intended in any way to limit the scope of the present invention. Relative terms such as “lower,” “upper,” “horizontal,” “vertical,” “above,” “below,” “up,” “down,” “top” and “bottom” as well as derivative thereof (e.g., “horizontally,” “downwardly,” “upwardly,” etc.) should be construed to refer to the orientation as then described or as shown in the drawing under discussion. These relative terms are for convenience of description only and do not require that the apparatus be constructed or operated in a particular orientation. Terms

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such as “attached,” “affixed,” “connected,” “coupled,” “interconnected,” and similar refer to a relationship wherein structures are secured or attached to one another either directly or indirectly through intervening structures, as well as both movable or rigid attachments or relationships, unless expressly described otherwise.

The term “action” is used herein in its conventional sense in the firearm art as meaning the mechanism that loads and ejects shells into/from the firearm and opens and closes the breech (i.e. the area in the receiver between an openable/closeable breech face on the front of the bolt and the rear face of the barrel chamber).

FIG. 1 is a longitudinal side elevation view of a firearm 20 having a recoil reduction system according to the present disclosure. The recoil reduction system may be a spring-type recoil suppression system in one embodiment, as further described herein. Firearm 20 includes a receiver 21, a trigger assembly 22 coupled to the receiver, a barrel 23 supported by the receiver, a forearm 24 extending forward from the receiver beneath the barrel for grasping by a user's hand, and buttstock 30 extending rearward from the receiver for placement against the user's shoulder when aiming the firearm held in a ready-to-fire position to acquire a target. The firearm 20 defines a longitudinal axis LA and axial direction coinciding with the centerline of the barrel 23 and its longitudinal bore formed therein (not shown). Firearm 20 may be any type of long gun, including a rifle or a shotgun.

Referring now to FIGS. 2-5 and 10, the buttstock 30 (or stock) in one embodiment is a substantially hollow and axially elongated structure having an opposing right sidewall 31a and left sidewall 31b, a front end 32 terminating at grip 34 and receiver mounting portion 33, an at least partially open rear end 35, a bottom 36, and a top 37. The sidewalls 31a, 31b define a longitudinally and axially extending internal cavity 38 for housing components of the recoil reduction system. In one embodiment, the rear end 35 may be at least partially open to access the cavity 38 and allow insertion of the recoil reduction system components shown in FIGS. 4A-B. The buttstock defines a stock axis SA which is offset from but parallel to longitudinal axis LA, both of which define axial directions.

A butt or recoil pad 40 is mounted on rear end of the buttstock. The butt pad 40 may be made of an energy absorbing resilient deformable material, such as for example without limitation foam, gel, rubber, or other material. In other embodiments the butt pad may be made of a rigid material (e.g. hard plastic, wood, etc.). A longitudinally extending cheek rest or comb 43 is mounted on the top 37 of the buttstock 30 for placement against a user's cheek when holding the firearm 20 in the ready-to-fire aiming position. In one embodiment, the comb 43 is fixedly attached to the buttstock 30 so that the comb moves rearward with the buttstock under recoil as a unit. Buttstock 30 may further include a removable bottom cover 30a which provides access for mounting the butt pad assembly to the buttstock, as further described herein.

Referring to FIGS. 2-5, the recoil reduction system is comprised of a “stationary” group of components and a “moving” group of components which are configured to interact under recoil to suppress and dampen the felt recoil experienced by a user. The terms “stationary” and “moving” refer to those components that either remain relatively immobile or are in motion respectively under recoil after the firearm is discharged.

The stationary components generally comprise the butt pad 40, hard insert 42, spacer plate 50, ram 60, skirt 70, skirt retaining plate 110, fasteners 41, and preload adjustment

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mechanism comprising a preload adjustment screw **80**, and transverse dowel pins **100**. Collectively, these components may be considered to form a "butt pad assembly" **40a** which removably mounts to the rear end of the buttstock **30**. In use, these butt pad components are buttressed by a user's shoulder and remain relatively stationary with respect to the moving components recognizing that the user's body will naturally undergo some slight motion and deformation due to the recoil forces generated.

Butt pad **40** may be overmolded onto hard insert **42** which provides a rigid structure or backbone for securing the more resilient pad to ram **60** via threaded fasteners **41** that extend through corresponding holes formed in the skirt retaining plate **110** and spacer plate **50** interspersed between the butt pad and ram. The fasteners **41** each engage a corresponding threaded socket **44** disposed in the ram. In some embodiments, the sockets **44** may be formed in threaded metallic inserts disposed in upper and lower longitudinal passages **44b**, **44a** formed in the ram **60**. In other embodiments, the longitudinal passages **44a**, **44b** may themselves instead be internally threaded to receive the threaded stem of the fasteners **41**. Either arrangement may be used. It bears noting that in certain embodiments where the butt pad **40** is formed of a relatively rigid material, the hard insert **42** may be omitted. Butt pad **40** may include through holes **45** used to access the heads of fasteners **41** for installing and removing the butt pad from the stock **30**. Accordingly, butt pad in such an embodiment is removably attached to the buttstock and readily replaceable.

Hard insert **42** may be made of a suitably rigid material, including without limitation hard plastic, metal (e.g. aluminum, titanium, steel, etc.), composites, etc. Butt pad **40** may have a vertically elongated oblong shape which is configured and contoured to abuttingly engage a user's shoulder. Hard insert **42** may have a similar complementary shape in some embodiments (see, e.g. FIG. 5). Hard insert **42** may be shorter in height than the buttstock **40** and not visible when the butt pad **40** is mounted to the buttstock **30**.

With continuing reference to FIGS. 2-5, the butt pad **40** is abuttingly engaged with the spacer plate **50** which may be in the form of a vertically oriented oblong straight plate in one embodiment. Spacer plate **50** may in turn abuttingly engage the skirt retaining plate **110** interspersed between the spacer and ram as illustrated. The skirt retaining plate **110** may have a vertically oblong shape similar to spacer plate **50**. The skirt retaining plate **110** preferably has a height substantially the same as the forward end portion of butt pad **40** as shown to create a relatively flush transition on the exterior surface of the buttstock transition. Skirt retaining plate **110** has a height slightly smaller than the spacer plate **50** and is not accessible or visible after assembly of the butt pad assembly **40a** to the buttstock **30** (see, e.g. FIG. 4A). The butt pad **40**, spacer plate **50**, and skirt retaining plate **110** are mutually configured and dimensioned to be generally similar and complementary in shape and dimension (height and width).

In one embodiment, the butt pad hard insert **42**, spacer plate **50**, and skirt retaining plate **110** are further mutually configured to form an interlock fit creating a nestable and stackable assembly. Referring to FIGS. 4A, 8A-C, 9A-C, and 10A-B, spacer plate **50** includes a raised front face **51** inset from the peripheral edges of the plate which is insertable into a mating recessed seat **111** formed in the rear surface of skirt retaining plate **110** forming one interlock feature. Another interlock feature is formed by a cylindrical pin **52** on the front face **51** of spacer plate **50** which is insertable through a mating hole **112** formed through the skirt retaining plate **110**. The skirt retaining plate **110**

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includes a cylindrical protrusion **113** which is insertable into a through bore **64** formed in the ram **60** (see also FIG. 6C) which forms an interlock feature and helps properly register the plate **110** in position with the ram.

In one embodiment to access preload adjustment screw **80**, protrusion **113** on skirt retaining plate **110** may include an axial through hole **114**. Through hole **114** communicates with corresponding vertically elongated through slot **55** in spacer plate **50**, elongated through slot **48** in hard insert **42**, and vertically elongated through passage **49** in butt pad **40**. The combination of through holes and slots are sufficient to provide a linear path for inserting the shaft of a hex key (not shown) into a mating hex socket formed in the enlarged head of the adjustment screw **80** without disassembling the butt pad assembly **40a** from the buttstock **30**.

With continuing reference to FIGS. 4A, 8A-C, 9A-C, and 10A-B, the rear face of spacer plate **50** includes a recessed seat **53** which receives a raised front face **46** inset from the peripheral edges of the hard insert **42** overmolded with butt pad **40** forming an interlock feature. The raised front face **46** projects forward beyond the peripheral portions of the butt pad **40** (see, e.g. FIG. 4A). Another interlock feature is formed by a cylindrical pin **47** on the front face **46** of insert **42** which is insertable into a mating circular socket **54** formed in the rear face of the spacer plate **50**. In one embodiment, socket **54** may be coaxially aligned with the pin **52** on the front face of the spacer plate **50**.

According to another aspect of the invention, the length of the buttstock **30** and butt pad assembly **40a** may be lengthened by stacking two or more spacer plates **50** together. The spacer plates **50** with foregoing front and rear face interlock features described above are configured to permit such stacking interlocked arrangement and assembly.

Referring now to FIGS. 4A and 6A-D, the ram **60** includes a vertically oriented mounting flange **61** at the rear against which skirt retaining plate **110** is mounted. In one embodiment, ram **60** further includes an axially elongated cantilevered mounting extension **62** and cylindrical plunger mounting protrusion **63** which is slidably received in tubular sleeve **7**. The mounting extension **62** projects in a forward direction from the flange **61** and is slideably coupled to the rear end **35** of buttstock **30**. This couples the entire assembly of the butt pad **40** with hard insert **42**, spacer **50**, face plate **110**, and ram **60** (butt pad assembly **40a**) to the buttstock **30**, thereby locating the assembly outboard and rearward of the rear end and main body of the buttstock (see, e.g. FIG. 4A). In one configuration, mounting extension **62** projects farther forward than plunger mounting protrusion **63** to securely mount and balance the butt pad assembly **40a** to the buttstock. Mounting extension **62** may have a polygonal configuration in one implementation; however, other shapes are possible for use.

In one embodiment, the ram **60** may be movably coupled to the rear end **35** of buttstock **30** via a pair of lateral dowel pins **100**. The dowel pins may extend transversely to the longitudinal axis LA and centerline of the buttstock through a pair of axially/horizontally elongated spaced apart slots **101** formed in the right and left sidewalls **31b**, **31a** of the buttstock **30** (see, e.g. FIG. 3). The pins **100** in turn pass through mating circular holes **65** in mounting extension **62**. This arrangement slideably couples the mounting extension **62** and butt pad assembly **40a** to the buttstock **30**. When the firearm **20** is in the ready-to-fire condition prior to discharge, the rearward biasing force of springs **90** and **91** urges the front and rear dowel pins **100** rearwards towards the rear ends of the slots, thereby locking the ram **60** in position on the buttstock **30**. It should be noted that the combination of

the dowel pins and their respective elongated slots allows the ram 60 to move linearly with respect to the buttstock 13, as further described herein.

Plunger mounting protrusion 63 includes axial through bore 64 which receives preload adjustment screw 80. The front stem end of screw 80 abuttingly engages a rear spring guide 130 which collectively forms a plunger. In one embodiment, the front end of the screw stem is not fixedly attached to the rear spring guide 130 through the open rear end 127 of the spring sleeve 120. The springs 90 and 91 bias the rear spring guide 130 into engagement with the adjustment screw.

The rear end of the adjustment screw 80 which includes the head is disposed in through bore 64 and accessible for adjusting the preload tension in the compression springs 90, 91 for firing either light or heavy ammunition shell loads which may be shotgun shells in one non-limiting example. Through bore 64 has a stepped configuration with a rear portion of the bore housing the head of the adjustment screw 80 having a larger diameter than the forward portion housing the stem of the screw. Internal threads are disposed in the smaller diameter forward portion of the through bore 64 for rotatably engaging the threaded stem of the screw 80 allowing the position of the plunger to be adjusted and varied by a user. In one non-limiting embodiment, the threads may be formed on a threaded insert 66 such as threaded nuts disposed in the through bore 64. In other embodiments, the front portion of the through bore 64 may be directly threaded instead.

To provide an audible and tactile confirmation of a full adjustment screw 80 rotation to a user, some embodiments may include a spring and ball detent 67. The detent is disposed transversely to the axial through bore 64 and positioned to engage the threaded stem of the adjustment screw 80. Rotating the screw a full 360 produces an audible “click” and tactile feedback sensation to the user when adjusting the preload mechanism.

In one embodiment, plunger mounting protrusion 63 may be connected to mounting extension 62 by an obliquely angled vertical support rib 68 extending between them. The rib 68 which mutually supports both of these cantilevered members and add rigidity to the structure. The rib 68 extends upwards from the top of mounting extension 62 to the plunger mounting protrusion 63. Rib 63 may be wedge or triangular shaped and its height diminishes moving rearward to forward.

The plunger mounting protrusion 63 is insertable into a tubular sleeve 120 disposed in the buttstock 30 which houses the spring assembly. Accordingly, protrusion 63 has a diameter which is sized slightly smaller than the inside diameter of sleeve 120 to allow forward and rearward reciprocating movement therein under recoil after discharging the firearm 20.

Ram 60 may further include an upper thrust block 61a and lower thrust block 61b formed on the front face or side of mounting flange 61. Thrust blocks 61a, 61b are raised structures projecting forward from flange 61 that define forward facing thrust surfaces arranged to engage skirt mounting plate 140 disposed on the rear end 35 of buttstock 30 under recoil (see, e.g. FIGS. 4A and 4B). The thrust blocks help evenly distribute the recoil forces to the mounting flange 61 and in turn the butt pad 40 to reduce the felt recoil by a user. Thrust block 61a may be disposed on each side of plunger mounting protrusion 63 in one configuration. Thrust block 61b may be disposed on each side and the bottom of the mounting extension 62. In one embodiment, the upper longitudinal passage 44a used for receiving a

threaded fastener to mount the butt pad 40 to the ram 60 may be formed in a barrel-shaped fastener protrusion 60a extending forward from the front side or face of the ram mounting flange 61 (see also FIGS. 6A, 6B, and 6D). The fastener protrusion 60a may be contiguous in structure with the upper thrust block 61a and have a front face which terminates in the same vertical plane as the upper thrust block face so that the fastener protrusion further serves as a thrust surface which distributes the recoil forces. In one embodiment, the lower longitudinal passage 44b which receives a threaded fastener may be formed in and through lower thrust block 61b.

Ram 60 may be made of any suitably strong material having sufficient rigidity to properly support the entire butt pad assembly 40a from the buttstock 30 in a movable manner. In one embodiment, ram 50 may be formed of a polymer for weight reduction, such as 33% glass filled nylon or other plastic. In other embodiments, ram 60 may be formed of a suitable metal such as aluminum, titanium, or other. In one embodiment, ram 60 including mounting extension 62 and plunger mounting protrusion 63 are integrally formed as a single unitary structure such as by molding or casting.

Referring to FIGS. 4A-B, 5, and 11A-B, the skirt 70 is a resiliently flexible tubular member interspersed between the stationary and moving components of the recoil reduction system. In the non-limiting embodiment shown, the skirt 70 is therefore fixedly mounted to both the stationary components and the moving components of the recoil reduction system. The rear end of skirt 70 may be attached to the rear flange of ram 60 and the front end of the skirt may be attached to the rear end 35 of the buttstock 30 as shown. Threaded fasteners may be used to connect skirt 70 to both the buttstock 30 and rear flange 61 of ram 60 in some embodiments. The rear end of skirt 5 may include an inwardly extending lip 72 which is trapped between the skirt retaining plate 110 and rear flange 61 of ram 60 as shown to mount the skirt to the stationary components (i.e. butt pad assembly 40a). Lip 72 is received in a complementary configured and arranged annular peripheral recess 69 formed on the rear face of the ram flange 61 (see, e.g. FIGS. 6B and 6C).

The front end of skirt 70 includes an inwardly extending lip 73 which is trapped between the rear end 35 of buttstock 30 and the skirt mounting plate 140. The plate 140 may be secured to the rear end of the buttstock 30 via threaded fasteners. Skirt retaining plate 140 includes a vertically elongated slot 141 of sufficient dimension to receive mounting extension 62 and plunger mounting protrusion 63 of the ram 60 therethrough for slidable movement inside the buttstock. In one configuration, the slot 141 may have a keyhole shape to complement the transverse cross-sectional shape of the mounting extension 62 and plunger mounting protrusion 63. In one implementation, skirt 70 may be overmolded onto skirt mounting plate 140 forming an integral assembly.

Other suitable securement means may be used for attaching skirt 70 to the buttstock 30 and ram 60 including for example adhesives, rivets, etc. Accordingly, the invention is not limited by the method of attachment used.

Skirt 70 defines a rearwardly open internal chamber 71 into which ram 60 may be inserted through the rear end of the skirt, as best shown in FIGS. 4A-B. The front top portion of the skirt may have a recessed area 74 which receives and complements the shape of the rear end of the comb 43 (see also FIG. 11A).

In one embodiment, the skirt **70** is made of a resiliently deformable elastomeric material having an elastic memory which may be temporarily deformed and then returns to its original shape. Skirt **70** serves as a flexible protective cover to enclose an axial gap **G** between the rear flange of ram **4** and rear end of the buttstock, thereby acting as a bellows or expansion joint therebetween. In one embodiment, without limitation, skirt **70** may be made of rubber. Other suitable elastomeric materials however may be used including for example various polymeric and thermoplastic elastomers suitable for the application.

Referring now to FIGS. 1-5, the moving components generally comprise comb **43**, spring sleeve **120**, front spring guide **132**, heavy spring **90**, light spring **91**, and buttstock **30**. The comb **43** may be removable and detachably mounted to the top of the buttstock **30**. In one embodiment, the buttstock **30** may be removably coupled to the receiver **21** via a threaded stock bolt **25** which engages a threaded socket formed in the rear of the receiver in a known manner. During recoil, the buttstock **30** therefore travels rearward with the receiver **21** and the barrel **23** (coupled to the front of the receiver) as a unit.

Referring also to FIGS. 7A-D, the spring sleeve **120** has a hollow cylindrical or tubular body which houses the spring mechanism (i.e. springs **90**, **91**) disposed therein in an axially elongated interior passage or receptacle **121**. In one embodiment, sleeve **120** has a partially closed front end **126** and an open rear end **127** through which the spring assembly **90**, **91** and plunger assembly (i.e. rear spring guide **130**, adjustment screw **80**, and plunger mounting protrusion **63**) may be slideably inserted. In one embodiment, the front end of sleeve **120** includes a forwardly open socket **125** in which a cylindrical protrusion or boss **128** of the buttstock **30** is at least partially received for positioning the sleeve. In one embodiment, the boss **128** may be frustoconical shaped and the socket **125** is complementary configured to have a mating frustoconical shape. In other possible embodiments, the boss and socket may have straight substantially parallel sides. A front wall **129** separates the socket **125** from the interior receptacle **121** except for a longitudinal passage **124** extending into the socket and through a cylindrical spring mounting protrusion **123** disposed in the interior receptacle. A cylindrical rod **123a** is received in the passage **124** (see, e.g. FIG. 4A) which forms part of the front spring guide **131**, as further described herein.

In one embodiment, the sleeve **120** is held in position in the buttstock **13** by operation of the spring assembly alone which biases the front wall of the sleeve into abutting engagement with the boss **128** as shown. In other possible embodiments, fasteners or pins may be used in addition to or instead of relying on the spring assembly secure the sleeve to the buttstock. Other ways of mounting the sleeve in the buttstock are possible.

Sleeve **120** has a diameter dimensioned to slidably receive the plunger assembly of ram **60** which may have a complementary cylindrical shape. Sleeve **120** may include a vertical flange **127a** on its rear end **127** disposed at an angle between 0 and 90 degrees to the cylindrical body of the sleeve. The flange **127a** is configured to engage a portion of the rear end **35** of the buttstock **30** such as skirt mounting plate **140** to properly position the rear end of the sleeve for receiving the ram plunger assembly (i.e. rear spring guide **130**, adjustment screw **80**, and plunger mounting protrusion **63**) into the sleeve. The flange **127a** may include a pair of opposing lateral cutouts **127b** which engage mating longitudinal protrusions **127c** formed in the sidewalls **31a**, **31b** and cavity **38** of the buttstock **30** (see also FIG. 12).

In one embodiment, a longitudinally extending slot **122** is formed in the bottom of the tubular spring sleeve **120** extending from the rear end **127** forward for a distance. The slot slideably receives the support rib **68** of the ram **60** thereby allowing for full movement of the plunger assembly inside the tube from the rear end.

The spring assembly includes an inner light spring **91**, outer heavy spring **90**, and opposing front and rear spring guides **131**, **130** (see FIGS. 4 and 5). In one embodiment, the springs are coiled compression springs. Other suitable type springs may be used. The inner and outer springs **131**, **130** are concentrically aligned and arranged with each other such that the inner spring nests inside the outer spring. The rear spring guide **130** may be cylindrical in shape and includes an enlarged head having a diameter slightly smaller than the inside diameter of the sleeve receptacle **121** and a stem of smaller diameter than and projecting perpendicular to the head. The stem may have a stepped configuration defining two shoulders on different diameter portions—a first shoulder **133** located between the head and larger diameter portion of the stem and a second shoulder **134** located between the larger diameter portion of the stem and a smaller diameter portion as shown. The first shoulder **133** is arranged to abuttingly engage the ends of the outer heavy spring **90** and the second shoulder **134** is arranged to engage the ends of the inner light spring **91**. The outer heavy spring **90** receives the larger diameter portion of the stem therein and the smaller diameter portion of the stem receives the inner light spring **91** therein.

The front spring guide **131** comprises rod **131** which is inserted into the front end of the light spring **91** and the spring mounting protrusion **123** of the spring sleeve **120** which is inserted into the front end of the heavy spring **90**. The rod **131** and protrusion **123** collectively form the front spring guide **131** which together provide the same configuration and profile as the rear spring guide **130**.

By utilizing the concentrically arranged pair of the lighter inner spring **91** and heavier outer spring **90** (referring to the spring force of each spring), the desired total spring force contributed by both springs is achieved in a more compact axial space of the sleeve. This allows the buttstock to be made shorter and more compact if desired. In other embodiments, a single spring may be provided if the desired spring force can be obtained with the space allocated in the buttstock for the spring assembly.

According to one aspect of the invention, a spring preload adjustment mechanism is provided which allows a user to change the spring preload to accommodate firing light or heavy shotshell loads which will each generate different magnitudes of recoil force. When firing a heavy load, the spring preload should preferably be increased to maximize the spring force produced and dampening effect of the springs **90**, **91** to counteract a larger recoil force. The preload force stored in the spring will act against the recoil force applied during firing. When a light load is fired which will produce a smaller recoil force, the spring preload should preferably be decreased so that a comparatively smaller spring force is produced to effectively dampen the lesser recoil force. If the preload and spring force is at a maximum when firing a light load, the dampening effect may be too little which can transfer a disproportionately larger amount of recoil force to the shoulder of the user. In one example, without limitation, the adjustment mechanism may be designed with approximately 125 lbs. of preload. Other suitable spring preloads may be used.

With continuing reference to **12**, the preload adjustment mechanism may comprise preload adjustment screw **80**

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which threadably engages the threaded through bore 64 disposed near the front of the cylindrical plunger mounting protrusion 63 of ram 60. The adjustment screw 80 which is linearly movable by rotating the screw in opposing direction allows the position of the rear spring guide 130 to be changed by a user with respect to the spring sleeve 120 prior to firing the firearm 20. This affects the degree to which the springs 90, 91 are either compresses or relaxed/expanded which corresponds to different spring preload conditions for matching the preload to the type of ammunition (light or heavy) being fired. The preload is infinitely adjustable to optimize and lessen the felt recoil experienced by the user.

In use, the preload may be increased for firing heavy loads by rotating the adjustment screw 80 in a first direction to advance the stem in an axially forward direction. This moves the rear spring guide 130 correspondingly forward closer to the front spring guide 131, thereby compressing the springs 90, 91. To decrease the preload for lighter loads, the adjustment screw is rotated in a second opposite direction to retract the stem in an axially rearward direction. This increases the distance between the front and rear spring guides 131, 130, thereby expanding or decompressing the springs. The adjustment screw 80 may therefore be linearly translated through and set at a plurality of possible preload adjustment positions or conditions to match the load type being fired. In one embodiment, through passage 49 in butt pad 40 allows the user to operate the preload adjustment screw with an elongated tool such as a hex key or screw driver without removing the butt pad from the buttstock 30 (see, e.g. FIG. 2).

A method for operating the recoil reduction system will now be described with general reference to FIGS. 1-5. The user may first optionally adjust the preload on the spring assembly in the manner described above to fit the type of load being fired (e.g. light or heavy). The buttstock is in a forward rest or inactive axial position (non-recoil position) shown in FIG. 4A. When the firearm is discharged, the buttstock and its group of "moving" components moves rearward with respect to the "stationary" group of components (see directional arrow in FIG. 4B). The recoil or butt pad 40, spacer plate 50, and ram 60 remain relatively immobile being buttressed against the user's shoulder. The buttstock 30 is at least partially guided via sliding engagement between the dowel pins 100 in the mounting extension 62 and the longitudinal slots 101 in the sidewalls of the buttstock. The dowel pins 100 each move forward in the slots 101 to a front position in the slots from a prior rear position before firing.

The spring sleeve 120 and front spring guide 131 move rearward with respect to the rear spring guide 130, adjustment screw 80, and ram plunger mounting protrusion 63 ("plunger assembly") which are stationary. The plunger assembly collectively moves farther forward and deeper into the sleeve 120 during the recoil event. The inner light and outer heavy springs 91, 90 become compressed between the axially spaced part and opposing spring guides 130, 131 to at least partially absorb and dampen the recoil force and felt recoil by the user. The resilient skirt 70 radially expands outwards as the rear end 35 of the buttstock 30 moves towards the flange 61 of the ram 60, thereby decreasing the gap formed therebetween. The upper and lower thrust blocks 61a, 61b abuttingly engage the rear end 35 (i.e. skirt mounting plate 140) of the buttstock 30 which arrests rearward movement of the buttstock assembly. This is shown in the rearward recoil position of the buttstock in FIG. 4B.

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When the buttstock has moved a maximum amount to the rearward active axial recoil position, the compressed spring assembly 90, 91 expands to its original pre-firing position thereby returning the buttstock 30 back to the inactive forward non-recoil position shown in FIG. 4A. This foregoing recoil cycle is repeated each time the firearm is fired.

While the foregoing description and drawings represent exemplary embodiments of the present disclosure, it will be understood that various additions, modifications and substitutions may be made therein without departing from the spirit and scope and range of equivalents of the accompanying claims. In particular, it will be clear to those skilled in the art that the present invention may be embodied in other forms, structures, arrangements, proportions, sizes, and with other elements, materials, and components, without departing from the spirit or essential characteristics thereof. In addition, numerous variations in the methods/processes described herein may be made within the scope of the present disclosure. One skilled in the art will further appreciate that the embodiments may be used with many modifications of structure, arrangement, proportions, sizes, materials, and components and otherwise, used in the practice of the disclosure, which are particularly adapted to specific environments and operative requirements without departing from the principles described herein. The presently disclosed embodiments are therefore to be considered in all respects as illustrative and not restrictive. The appended claims should be construed broadly, to include other variants and embodiments of the disclosure, which may be made by those skilled in the art without departing from the scope and range of equivalents.

What is claimed is:

1. An adjustable recoil reduction system for a firearm, the system comprising:

an axially extending buttstock having a longitudinal stock axis, a rear end, a front end, and a longitudinally extending internal cavity;

a ram including a cantilevered mounting extension, the mounting extension projecting forward from the ram through the rear end of the buttstock into the internal cavity, the mounting extension slideably coupling the ram to the buttstock;

a butt pad coupled to the ram;

an adjustable plunger assembly comprising a rear spring guide disposed in the buttstock and a preload adjustment screw rotatably coupled to the ram and engaging the rear spring guide, the plunger assembly being movable forward and rearward in the buttstock; and a first compression spring mounted inside the buttstock and engaged by the rear spring guide, the plunger assembly operating to compress the first compression spring when the buttstock moves rearward relative to the plunger assembly;

wherein rotating the adjustment screw in a first direction advances the rear spring guide and compresses the first compression spring for setting a first preload condition, and rotating the adjustment screw in an opposite second direction retracts the rear spring guide and allows the first compression spring to expand for setting a second preload condition;

wherein when a rearward acting recoil force is generated by discharging the firearm, the buttstock moves rearward and compresses the first compression spring against the plunger assembly to absorb at least a portion of the recoil force.

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2. The system according to claim 1, wherein the assembly is inserted deeper into the buttstock when the buttstock moves rearward than when the buttstock is in a forward non-recoil position.

3. The system according to claim 1, further comprising a tubular spring sleeve disposed inside the buttstock which holds the first compression spring therein, the plunger assembly movably inserted in the spring sleeve.

4. The system according to claim 3, wherein a front end of the spring sleeve includes a socket engaged with a cylindrical boss formed in the internal cavity of the buttstock.

5. The system according to claim 1, further comprising an elastomeric skirt coupled to the rear end of the buttstock and the ram, the skirt compressing and radially expanding when the buttstock moves rearward.

6. The system according to claim 1, further comprising a second compression spring concentrically arranged around the first compression spring, the rear spring guide compressing both the first and second compression springs when the buttstock moves rearward.

7. The system according to claim 5, further comprising a front spring guide, the front and rear spring guides each including a first shoulder engaging the first compression spring and a second shoulder engaging the second compression spring.

8. The system according to claim 1, wherein the ram includes a plunger mounting protrusion comprising internal threads which rotatably engage the preload adjustment screw.

9. The system according to claim 1, further comprising a vertically elongated spacer plate disposed between the butt pad and the ram.

10. The system according to claim 1, wherein the butt pad includes a through passage that provides access to the preload adjustment screw without uncoupling the butt pad from the ram.

11. The system according to claim 1, wherein the buttstock includes an axially spaced apart pair of longitudinal slots each of which slideably engage a transverse dowel pin extending through the mounting extension of the ram to slideably couple the mounting extension to the buttstock.

12. An adjustable recoil reduction system for a firearm, the system comprising:

a buttstock extending rearward from a receiver, the buttstock having a longitudinal stock axis, a rear end, a front end, and an internal cavity extending between the front and rear ends;

a spring assembly disposed in the buttstock, the spring assembly comprising a tubular sleeve fixedly mounted inside the buttstock, a first spring inside the sleeve, a second spring inside the sleeve concentrically arranged around the first spring, a rear spring guide engaged with the first and second springs, and a front spring guide engaged with the first and second springs;

a butt pad assembly comprising a butt pad configured for placement against a shoulder of a user and a ram coupled to butt pad, the butt pad assembly movably coupled to the buttstock;

an axially elongated preload adjustment screw threadably coupled to the ram and having a front end engaging the rear spring guide to fix a position of the rear spring guide relative to the sleeve, the preload adjustment screw movable forward and rearward relative to the buttstock by rotating the preload adjustment screw in opposing directions;

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wherein rotating the preload adjustment screw in a first direction compresses the first and second springs to set a first preload condition, and rotating the preload adjustment screw in an opposite second direction expands the first and second springs to set a second preload condition;

wherein when a rearward acting recoil force is generated by discharging the firearm, the buttstock moves rearward and the rear spring guide compresses the first and second springs thereby absorbing at least a portion of the recoil force.

13. The system according to claim 12, wherein the first and second springs expand when the recoil force is removed via the buttstock moving forward.

14. The system according to claim 12, wherein the butt pad assembly remains stationary against the user's shoulder when the firearm is discharged and the buttstock and spring assembly move rearward.

15. The system according to claim 12, wherein the rear spring guide slides forward in the sleeve under the rearward acting recoil force.

16. The system according to claim 12, wherein the ram includes a vertically oriented mounting flange having a height substantially coextensive with a height of a front end of the butt pad.

17. The system according to claim 12, further comprising a spacer plate interspersed between the butt pad and the ram.

18. The system according to claim 12, wherein the buttstock further includes a sidewall and an axially elongated slot disposed in the sidewall, and further comprising a transverse pin mounted inside the buttstock inserted through the slot and connected to a forward projecting mounting extension disposed on the ram, the pin and slot acting to movably couple the butt pad assembly to the buttstock.

19. The system according to claim 12, further comprising an elastomeric skirt coupled to the rear end of the buttstock and the ram, the skirt compressing and radially expanding when the buttstock moves rearward.

20. The system according to claim 12, further comprising a vertically elongated spacer plate disposed between the butt pad and the ram.

21. The system according to claim 12, wherein the ram includes a thrust block which engages a vertically oblong plate attached to the rear end of the buttstock when the buttstock moves rearward under recoil.

22. A method for reducing recoil in a firearm, the method comprising:

providing a firearm including a buttstock, a butt pad assembly movably coupled to a rear end of the buttstock, a coiled first spring disposed inside the buttstock, a second spring concentrically arranged around the first spring, and an adjustable plunger mechanism comprising a rear spring guide engaging the compression spring and a preload adjustment screw rotatably coupled to the butt pad assembly and engaging the rear spring guide, the buttstock being in a forward non-recoil position spaced apart from the butt pad assembly by a first distance;

rotating the preload adjustment screw in a first direction which advances the rear spring guide forward in the buttstock;

compressing the first spring by the advancement of the rear spring guide to a first preload condition;

discharging the firearm;

moving the buttstock in a rearward direction under recoil closer to the butt pad, the buttstock being in a rearward

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recoil position spaced apart from the butt pad assembly  
by a second distance smaller than the first distance;  
the buttstock compressing the first spring by movement in  
the rearward direction;

expanding the first spring; and

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returning the buttstock to the forward non-recoil position;

wherein the step of compressing the first spring also  
compresses the second spring.

**23.** The method according to claim **22**, further comprising  
rotating the preload adjustment screw in a second direction 10  
which retracts the rear spring guide and expands the first  
spring.

\* \* \* \* \*

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